

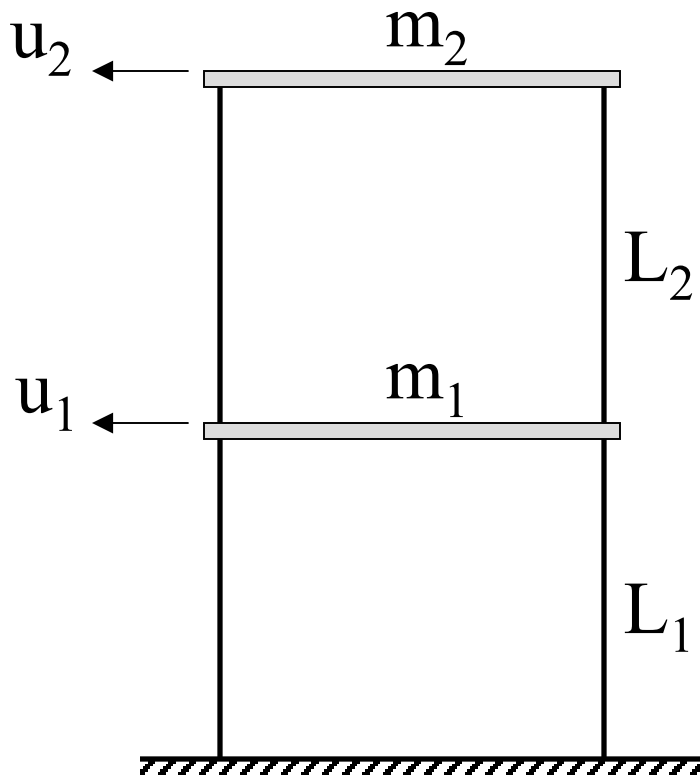
November 5, 2002

SE 180: Earthquake Engineering

# SE 180

# Final Project

# 2 Story Shear Frame



Given:

$m_1$

$L_1$

$L_2$

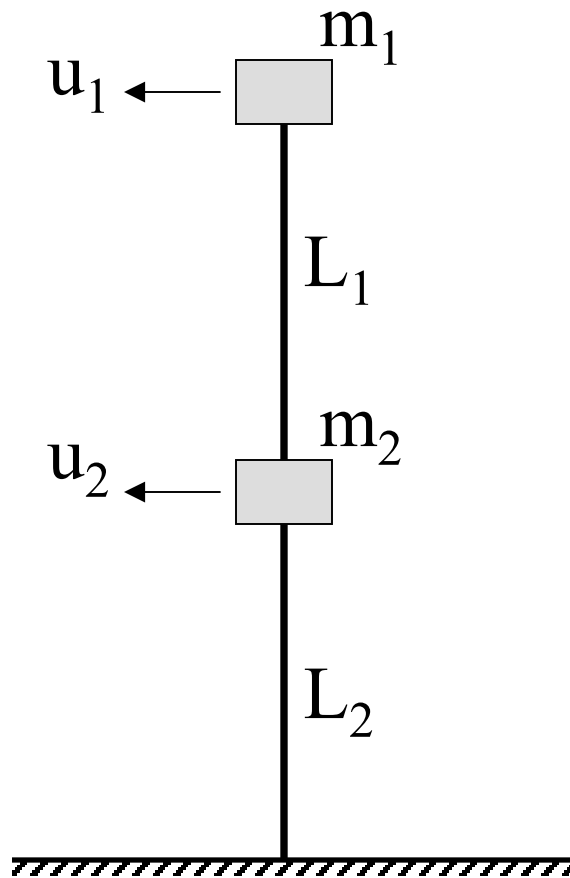
$EI$

$\omega_1$

$\omega_2$

Solve for  $m_2$

# 2 Story Bending Beam



Given:

$m_1$

$L_1$

$L_2$

$EI$

$\omega_1$

$\omega_2$

Solve for  $m_2$

# 3 Story Shear Frame

Given:

$m_1$

$m_2$

$L_1$

$L_2$

$L_3$

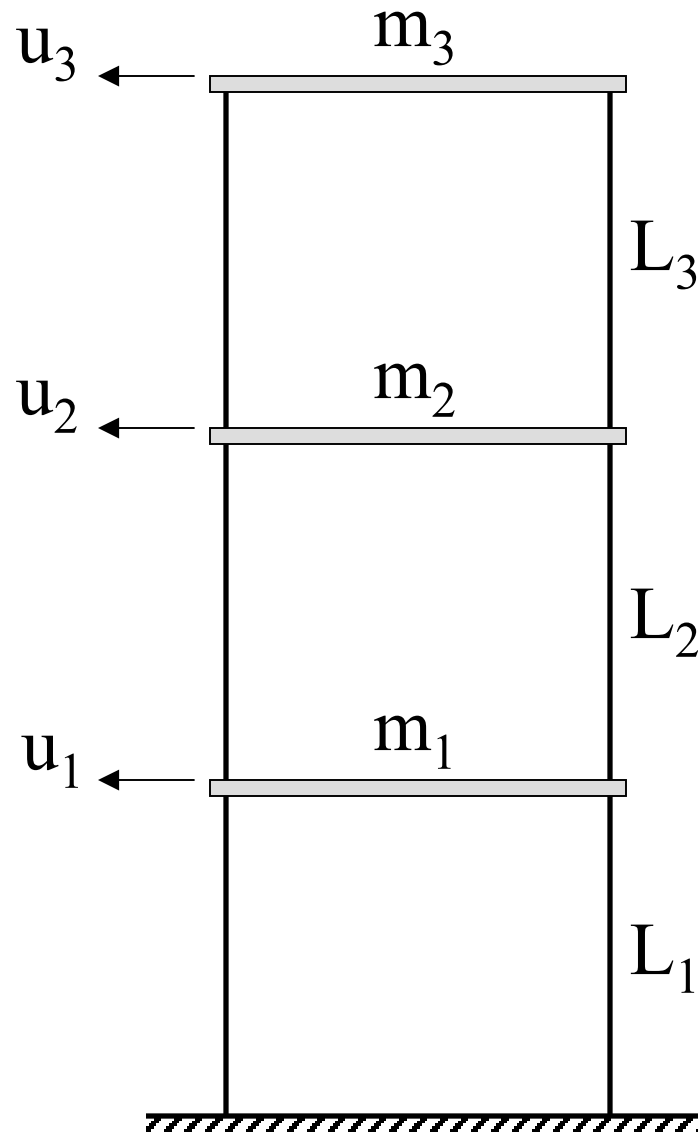
$EI$

$\omega_1$

$\omega_2$

$\omega_3$

Solve for  $m_3$



# 3 Story Bending Beam

Given:

$m_1$

$m_2$

$L_1=L_2=L_3=L$

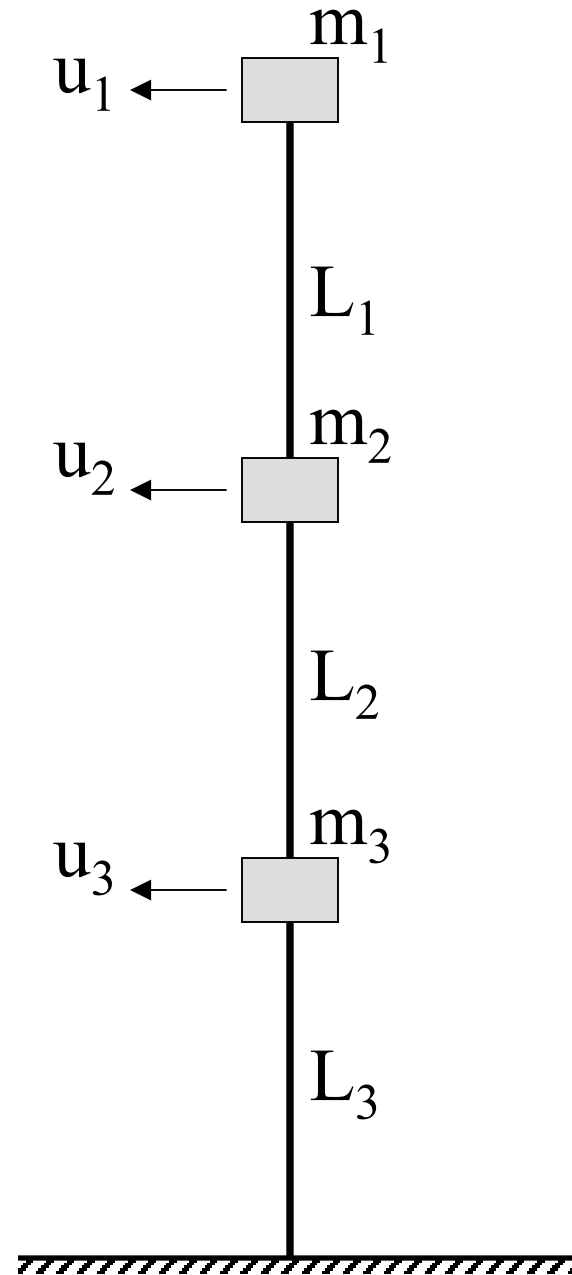
$EI$

$\omega_1$

$\omega_2$

$\omega_3$

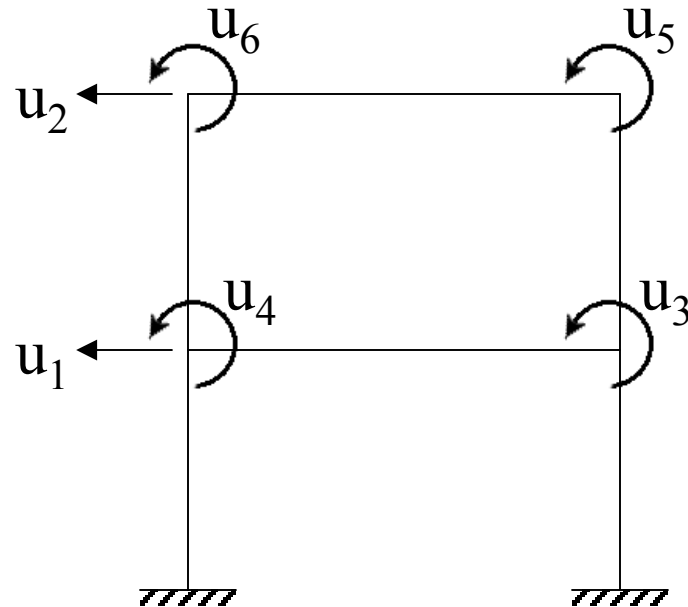
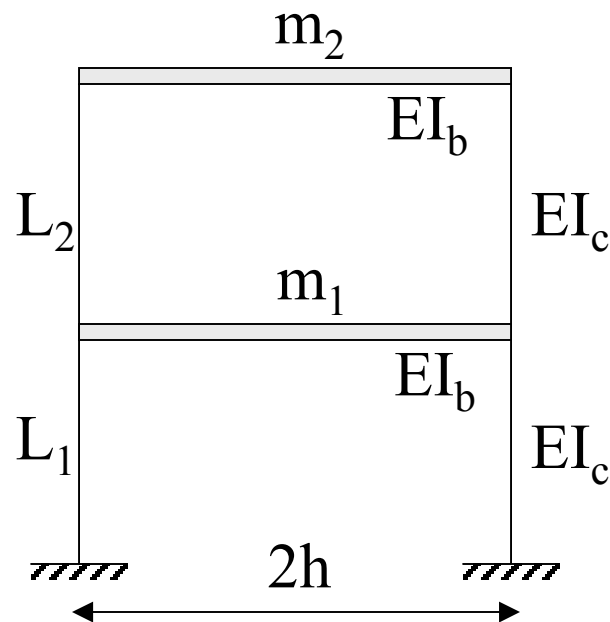
Solve for  $m_3$



## Part 1: Determining the unknown mass of your structure

Step 1: Assemble the mass and stiffness matrices for your structure

For Example: Find  $m_2$ , given  $m_1$ ,  $L_1 = L_2 = h$ ,  $EI_c$ , &  $EI_b$



Example (continued):

$$m = \begin{bmatrix} m_1 & 0 & 0 & 0 & 0 & 0 \\ 0 & m_2 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$k = \begin{bmatrix} \frac{48 \cdot E \cdot I_c}{h^3} & \frac{-24 \cdot E \cdot I_c}{h^3} & 0 & 0 & \frac{6 \cdot E \cdot I_c}{h^2} & \frac{6 \cdot E \cdot I_c}{h^2} \\ \frac{-24 \cdot E \cdot I_c}{h^3} & \frac{24 \cdot E \cdot I_c}{h^3} & \frac{-6 \cdot E \cdot I_c}{h^2} & \frac{-6 \cdot E \cdot I_c}{h^2} & \frac{-6 \cdot E \cdot I_c}{h^2} & \frac{-6 \cdot E \cdot I_c}{h^2} \\ 0 & \frac{-6 \cdot E \cdot I_c}{h^2} & \frac{8 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} & \frac{2 \cdot E \cdot I_c}{h} & 0 \\ 0 & \frac{-6 \cdot E \cdot I_c}{h^2} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} & \frac{8 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} & 0 & \frac{2 \cdot E \cdot I_c}{h} \\ \frac{6 \cdot E \cdot I_c}{h^2} & \frac{-6 \cdot E \cdot I_c}{h^2} & \frac{2 \cdot E \cdot I_c}{h} & 0 & \frac{4 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} \\ \frac{6 \cdot E \cdot I_c}{h^2} & \frac{-6 \cdot E \cdot I_c}{h^2} & 0 & \frac{2 \cdot E \cdot I_c}{h} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} & \frac{4 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} \end{bmatrix}$$

## Determining the unknown mass of your structure

Step 2: Perform Static Condensation (if necessary)

$$[k] = \begin{bmatrix} k_{tt} & k_{t0} \\ k_{0t} & k_{00} \end{bmatrix}$$

$$\hat{k}_{tt} = k_{tt} - k_{0t}^T k_{00}^{-1} k_{0t}$$

Example (continued):

$$k_{tt} = \begin{bmatrix} \frac{48 \cdot E \cdot I_c}{h^3} & \frac{-24 \cdot E \cdot I_c}{h^3} \\ \frac{-24 \cdot E \cdot I_c}{h^3} & \frac{24 \cdot E \cdot I_c}{h^3} \end{bmatrix}$$

$$k_{0t} = \begin{bmatrix} 0 & -\frac{6 \cdot E \cdot I_c}{h^2} \\ 0 & -\frac{6 \cdot E \cdot I_c}{h^2} \\ \frac{6 \cdot E \cdot I_c}{h^2} & -\frac{6 \cdot E \cdot I_c}{h^2} \\ \frac{6 \cdot E \cdot I_c}{h^2} & -\frac{6 \cdot E \cdot I_c}{h^2} \end{bmatrix}$$

$$k_{00} = \begin{bmatrix} \frac{8 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} & \frac{2 \cdot E \cdot I_c}{h} & 0 \\ \frac{2 \cdot E \cdot I_b}{2 \cdot h} & \frac{8 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} & 0 & \frac{2 \cdot E \cdot I_c}{h} \\ \frac{2 \cdot E \cdot I_c}{h} & 0 & \frac{4 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} \\ 0 & \frac{2 \cdot E \cdot I_c}{h} & \frac{2 \cdot E \cdot I_b}{2 \cdot h} & \frac{4 \cdot E \cdot I_c}{h} + \frac{4 \cdot E \cdot I_b}{2 \cdot h} \end{bmatrix}$$



## Step 2: Static Condensation (continued)

Example (continued):

$$\hat{\mathbf{k}}_{tt} = \mathbf{k}_{tt} - \mathbf{k}_{ot}^T \mathbf{k}_{oo}^{-1} \mathbf{k}_{ot} = \begin{bmatrix} 24 \cdot E \cdot I_c \cdot \frac{(32 \cdot I_c^2 + 63 \cdot I_c \cdot I_b + 18 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} & -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} \\ -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} & 24 \cdot E \cdot I_c \cdot \frac{(4 \cdot I_c^2 + 18 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} \end{bmatrix}$$

Where  $\hat{\mathbf{k}}_{tt}$  is the condensed stiffness matrix

## Determining the unknown mass of your structure

Step 3: Solve the Eigen-value problem to determine the determinant of

$$\left[ [k] - \omega^2 [m] \right]$$

Example (continued):

$$[k] - \omega^2 [m] = \begin{bmatrix} 24 \cdot E \cdot I_c \cdot \frac{(32 \cdot I_c^2 + 63 \cdot I_c \cdot I_b + 18 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} - \omega^2 \cdot m_1 & -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} \\ -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} & 24 \cdot E \cdot I_c \cdot \frac{(4 \cdot I_c^2 + 18 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} - \omega^2 \cdot m_2 \end{bmatrix}$$

Example (continued):

$$\left| \left[ [k] - \omega^2 [m] \right] \right| =$$

$$\frac{(576 E^2 I_c^4 + 5184 I_c^3 E^2 I_b - 96 I_c^3 \omega^2 m_1 h^3 E - 768 I_c^3 E \omega^2 m_2 h^3 + 28 I_c^2 \omega^4 m_1 h^6 m_2 - 432 I_c^2 \omega^2 m_1 h^3 E I_b + 5184 I_c^2 E^2 I_b^2 - 1512 I_c^2 E \omega^2 m_2 h^3 I_b - 432 I_c E \omega^2 m_2 h^3 I_b^2 - 216 I_c \omega^2 m_1 h^3 E I_b^2 + 36 I_c \omega^4 m_1 h^6 m_2 I_b + 9 \omega^4 m_1 h^6 m_2 I_b^2)}{[h^6 (28 I_c^2 + 36 I_c I_b + 9 I_b^2)]}$$

## Determining the unknown mass of your structure

Step 4: Plug the given values for  $m_1$ ,  $L_1$ ,  $L_2$ ,  $EI$ , &  $\omega_1$  into

$$\left| \left[ [k] - \omega^2 [m] \right] \right| = 0, \text{ and solve for the unknown mass.}$$

Example (continued):

Given:

$$m_1 = 0.2 \text{ kg}$$

$$L_1 = 0.3048 \text{ m}$$

$$L_2 = 0.3048 \text{ m}$$

$$EI = 1.0889242 \text{ n-m}^2$$

$$\omega_1 = 12.566 \text{ rad/s}$$

$$\omega_2 = 86.013 \text{ rad/s}$$

# Step 4: Solve for unknown m (continued)

Example (continued):

Plug  $m_1$ ,  $L_1$ ,  $L_2$ ,  $EI$ , &  $\omega_1$  into  $\left| \left[ [k] - \omega^2 [m] \right] \right| = 0$

Solve for  $m_2$

$$m_2 = 24 \cdot E \cdot I_c \cdot \frac{\left( 24 \cdot E \cdot I_c^3 + 216 \cdot I_c^2 \cdot E \cdot I_b - 4 \cdot I_c^2 \cdot \omega_1^2 \cdot m_1 \cdot h^3 - 18 \cdot I_c \cdot \omega_1^2 \cdot m_1 \cdot h^3 \cdot I_b + 216 \cdot I_c \cdot E \cdot I_b^2 - 9 \cdot \omega_1^2 \cdot m_1 \cdot h^3 \cdot I_b^2 \right)}{\left[ h^3 \cdot \omega_1^2 \cdot \left( 768 \cdot E \cdot I_c^3 - 28 \cdot I_c^2 \cdot \omega_1^2 \cdot m_1 \cdot h^3 + 1512 \cdot I_c^2 \cdot E \cdot I_b + 432 \cdot I_c \cdot E \cdot I_b^2 - 36 \cdot I_c \cdot \omega_1^2 \cdot m_1 \cdot h^3 \cdot I_b - 9 \cdot \omega_1^2 \cdot m_1 \cdot h^3 \cdot I_b^2 \right) \right]}$$

$$m_2 = 0.949 \text{ kg}$$

## Determining the unknown mass of your structure

Step 5: Perform a self-check on the value of the mass you just found

Plug the newly determined mass along with the remaining

$\omega_i$  into  $\left| \left[ [k] - \omega^2 [m] \right] \right|$  and verify that the determinant

is still equal to zero

Example (continued):

$$m_2 = 24 \cdot E \cdot I_c \cdot \frac{\left( 24 \cdot E \cdot I_c^3 + 216 \cdot I_c^2 \cdot E \cdot I_b - 4 \cdot I_c^2 \cdot \omega^2 \cdot m_1 \cdot h^3 - 18 \cdot I_c \cdot \omega^2 \cdot m_1 \cdot h^3 \cdot I_b + 216 \cdot I_c \cdot E \cdot I_b^2 - 9 \cdot \omega^2 \cdot m_1 \cdot h^3 \cdot I_b^2 \right)}{\left[ h^3 \cdot \omega^2 \cdot \left( 768 \cdot E \cdot I_c^3 - 28 \cdot I_c^2 \cdot \omega^2 \cdot m_1 \cdot h^3 + 1512 \cdot I_c^2 \cdot E \cdot I_b + 432 \cdot I_c \cdot E \cdot I_b^2 - 36 \cdot I_c \cdot \omega^2 \cdot m_1 \cdot h^3 \cdot I_b - 9 \cdot \omega^2 \cdot m_1 \cdot h^3 \cdot I_b^2 \right) \right]}$$

$$m_2 = 0.949 \text{ kg (same as with } \omega_1 - \text{OK!)}$$

## Part 2: Modal Analysis

Step 1: Determine the Mode Shapes  $\phi_n$

$$(\mathbf{k} - \omega^2 \mathbf{m}) \mathbf{u} = \mathbf{0}$$

Equation 1

Continuing with the Eigen-value problem solution (again, Matlab does this, or by hand for a 2-dof system), for each  $\omega_n$  we get an associated  $\phi_n \leftarrow$  mode shape. To do this (for each identified  $\omega_n$ ), go ahead and substitute this  $\omega_n$  for  $\omega$  in Eq. 1 above. Upon this substitution, you can solve for the corresponding vector  $\mathbf{u}$ , the components of which defines the mode shape  $\phi_n$ .

For the Previous example:

$$\omega_1 = 12.566 \text{ rad/s}$$

$$\begin{bmatrix} 24 \cdot E \cdot I_c \cdot \frac{(32 \cdot I_c^2 + 63 \cdot I_c \cdot I_b + 18 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} - \omega^2 \cdot m_1 & -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} \\ -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} & 24 \cdot E \cdot I_c \cdot \frac{(4 \cdot I_c^2 + 18 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} - \omega^2 \cdot m_2 \end{bmatrix} = \begin{bmatrix} 1.397 \cdot 10^3 & -581.566 \\ -581.566 & 242.065 \end{bmatrix}$$

$$\begin{bmatrix} 1.397 \cdot 10^3 & -581.566 \\ -581.566 & 242.065 \end{bmatrix} \cdot \begin{bmatrix} \phi_{11} \\ \phi_{21} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

Let  $\phi_{11} = 1.0$ ; therefore,  $\phi_{21} = 2.402$



Example (continued):

$$\omega_2 = 86.011 \text{ rad/s}$$

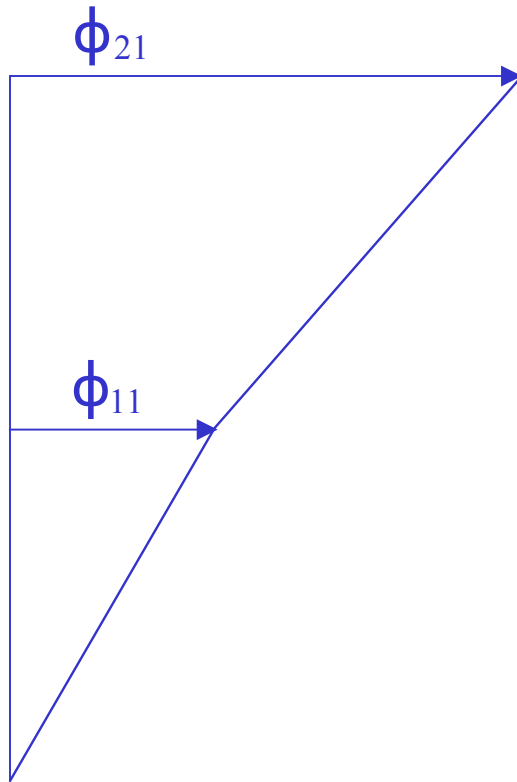
$$\begin{bmatrix} 24 \cdot E \cdot I_c \cdot \frac{(32 \cdot I_c^2 + 63 \cdot I_c \cdot I_b + 18 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} - \omega^2 \cdot m_1 & -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} \\ -24 \cdot E \cdot I_c \cdot \frac{(10 \cdot I_c^2 + 27 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} & 24 \cdot E \cdot I_c \cdot \frac{(4 \cdot I_c^2 + 18 \cdot I_c \cdot I_b + 9 \cdot I_b^2)}{[(28 \cdot I_c^2 + 36 \cdot I_c \cdot I_b + 9 \cdot I_b^2) \cdot h^3]} - \omega^2 \cdot m_2 \end{bmatrix} = \begin{bmatrix} -50.933 & -581.566 \\ -581.566 & -6.629 \cdot 10^3 \end{bmatrix}$$

$$\begin{bmatrix} -50.933 & -581.566 \\ -581.566 & -6.629 \cdot 10^3 \end{bmatrix} \cdot \begin{bmatrix} \phi_{12} \\ \phi_{22} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

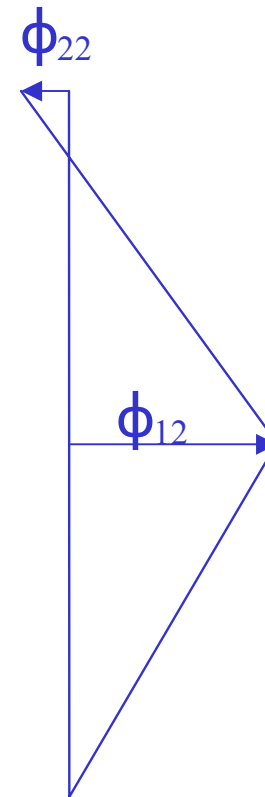
Let  $\phi_{12} = 1.0$ ; therefore,  $\phi_{22} = -0.0876$

Example (continued):

First Mode



Second Mode



## Part 2: Modal Analysis

Step 2: Determine the Modal Participation Factors  $\frac{L_i}{M_i}$

$$M_i = \sum_{j=1}^{\text{NDOF}} m_j \phi_{ji}^2$$

$$L_i = \sum_{j=1}^{\text{NDOF}} m_j \phi_{ji}$$

$$\phi_i = \begin{bmatrix} \phi_{1i} \\ \phi_{2i} \\ \cdot \\ \cdot \\ \cdot \\ \phi_{\text{NDOF } i} \end{bmatrix}$$

Example (continued):

$$M_i = \sum_{j=1}^{\text{NDOF}} m_j \phi_{ji}^2$$

$$M_1 = m_1 \phi_{11}^2 + m_2 \phi_{21}^2 = (0.2)(1)^2 + (0.949)(2.402)^2 = 5.67535 \text{ kg}$$

$$M_2 = m_1 \phi_{12}^2 + m_2 \phi_{22}^2 = (0.2)(1)^2 + (0.949)(-0.0876)^2 = 0.20728 \text{ kg}$$

$$L_i = \sum_{j=1}^{\text{NDOF}} m_j \phi_{ji}$$

$$L_1 = m_1 \phi_{11} + m_2 \phi_{21} = (0.2)(1) + (0.949)(2.402) = 2.4795 \text{ kg}$$

$$L_2 = m_1 \phi_{12} + m_2 \phi_{22} = (0.2)(1) + (0.949)(-0.0876) = 0.11687 \text{ kg}$$

Example (continued):

$$\frac{L_1}{M_1} = \frac{2.4795 \text{ kg}}{5.67535 \text{ kg}} = 0.437$$

$$\frac{L_2}{M_2} = \frac{0.11687 \text{ kg}}{0.20728 \text{ kg}} = 0.563$$

## Part 2: Modal Analysis

Step 3: Determine  $K_i$

$$K_i = \omega_i^2 M_i$$

$$K_1 = \omega_1^2 M_1 = (12.566)^2 (5.67535) = 896.1625$$

$$K_2 = \omega_2^2 M_2 = (86.011)^2 (0.20728) = 1533.435$$

## Part 2: Modal Analysis

### Step 4: Add Damping

Now, you can add any modal damping you wish (which is another big plus, since you control the damping in each mode individually). If you choose  $\zeta_i = 0.02$  or  $0.05$ , the equations become:

$$\ddot{q}_i + 2\xi_i\omega_i\dot{q}_i + \omega_i^2q_i = -\frac{L_i}{M_i}\ddot{u}_g, \quad i = 1, 2, \dots \text{NDOF}$$

## Part 2: Modal Analysis

### Step 5: Solve for $q_i(t)$

Solve for  $q_i(t)$  in the above uncoupled equations (using a SDOF-type program), and the final solution is obtained from:

$$\mathbf{u} = \Phi \mathbf{q}$$

$$\dot{\mathbf{u}} = \Phi \dot{\mathbf{q}}$$

$$\ddot{\mathbf{u}} = \Phi \ddot{\mathbf{q}}$$

$$\ddot{\mathbf{u}}^t = \ddot{\mathbf{u}} + \mathbf{1} \ddot{u}_g$$



We will solve for  $q_i(t)$  using a modified version of the spreadsheet for solving for the response of a SDOF system using Newmark's Method

## **Part 3: Spreadsheet for Modal Analysis**

### **Step-By-Step Procedure For Setting Up A Spreadsheet For Using Newmark's Method and Modal Analysis To Solve For The Response Of A Multi-Degree Of Freedom (MDOF) System**

Start with the equation of motion for a linear multi-degree of freedom system with base ground excitation:

$$\mathbf{m}\ddot{\mathbf{u}} + \mathbf{c}\dot{\mathbf{u}} + \mathbf{k}\mathbf{u} = -\mathbf{m}\mathbf{1}\ddot{u}_g$$

Using Modal Analysis, we can rewrite the original coupled matrix equation of motion as a set of un-coupled equations.

$$\ddot{q}_i + 2\zeta\omega\dot{q}_i + \omega_i^2 q_i = -\frac{L_i}{M_i} \ddot{u}_g, \quad i = 1, 2, \dots, \text{NDOF}$$

with initial conditions of  $d_i(t=0) = d_{i_0}$  and  $v_i(t=0) = v_{i_0}$

Note that total acceleration or absolute acceleration will be

$$\ddot{q}_{i \text{ abs}} = \ddot{q}_i + \ddot{u}_g$$

We can solve each one separately (as a SDOF system), and compute histories of  $q_i$  and their time derivatives. To compute the system response, plug the  $q$  vector back into  $\mathbf{u} = \Phi \mathbf{q}$  and get the  $u$  vector (and the same for the time derivatives to get velocity and acceleration).

The beauty here is that there is no matrix operations involved, since the matrix equation of motion has become a set of un-coupled equation, each including only one generalized coordinate  $q_n$  .

In the spreadsheet, we will solve each mode in a separate worksheet.

## *Step 1 - Define System Properties and Initial Conditions for First Mode*

(A) Begin by setting up the cells for the Mass, Stiffness, and Damping of the SDOF System (Fig. 1). These values are known.

(B) Set up the cells for the modal participation factor  $\frac{L_i}{M_i}$  and mode shape  $\phi_i$  (Fig. 1). These values must be determined in advance using Modal Analysis.

(C) Calculate the Natural Frequency of the SDOF system using the equation

$$\omega_i = \sqrt{K_i/M_i} \quad (\text{Equation 1})$$

*Note:* If the system damping is given in terms of the Modal Damping Ratio ( $\zeta_i$ ) then the Damping ( $C_i$ ) can be calculated using the equation:

$$C_i = 2 \zeta_i \omega_i M_i \quad (\text{Equation 2})$$

(D) Set up the cells for the 2 Newmark Coefficients  $\alpha$  &  $\beta$  (Fig. 1), which will allow for performing

a) the Average Acceleration Method, use  $\alpha = \frac{1}{2}$  and  $\beta = \frac{1}{6}$  .

b) the Linear Acceleration Method, use  $\alpha = \frac{1}{2}$  and  $\beta = \frac{1}{4}$  .

(E) Set up cells (Fig. 1) for the initial displacement and velocity ( $d_o$  and  $v_o$  respectively)

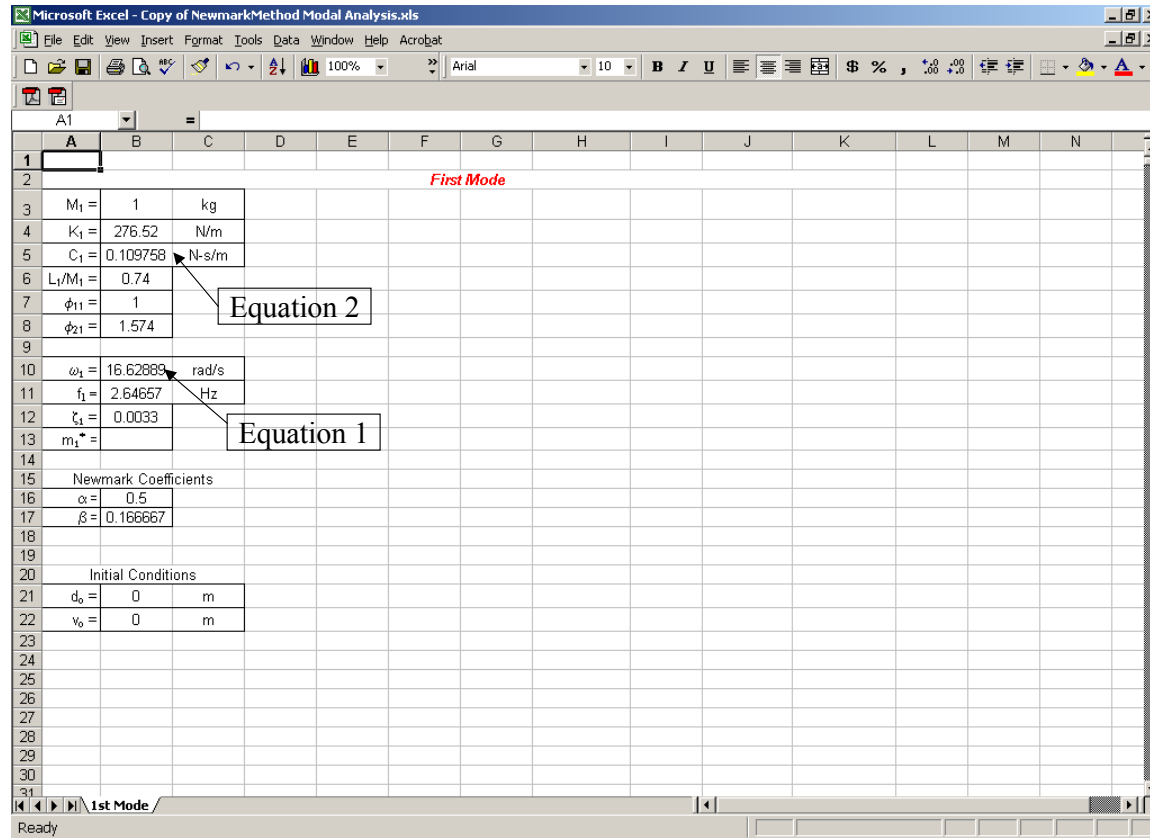


Figure 1: Spreadsheet After Completing Step 1

*Step 2 – Set Up Columns for Solving The Equation of Motion Using  
Newmark's Method*

Place a cell (Fig. 2) for the time increment ( $\Delta t$ ).

Place columns (Fig. 2) for the time, base excitation, applied force divided by mass, relative acceleration, relative velocity, and relative displacement.



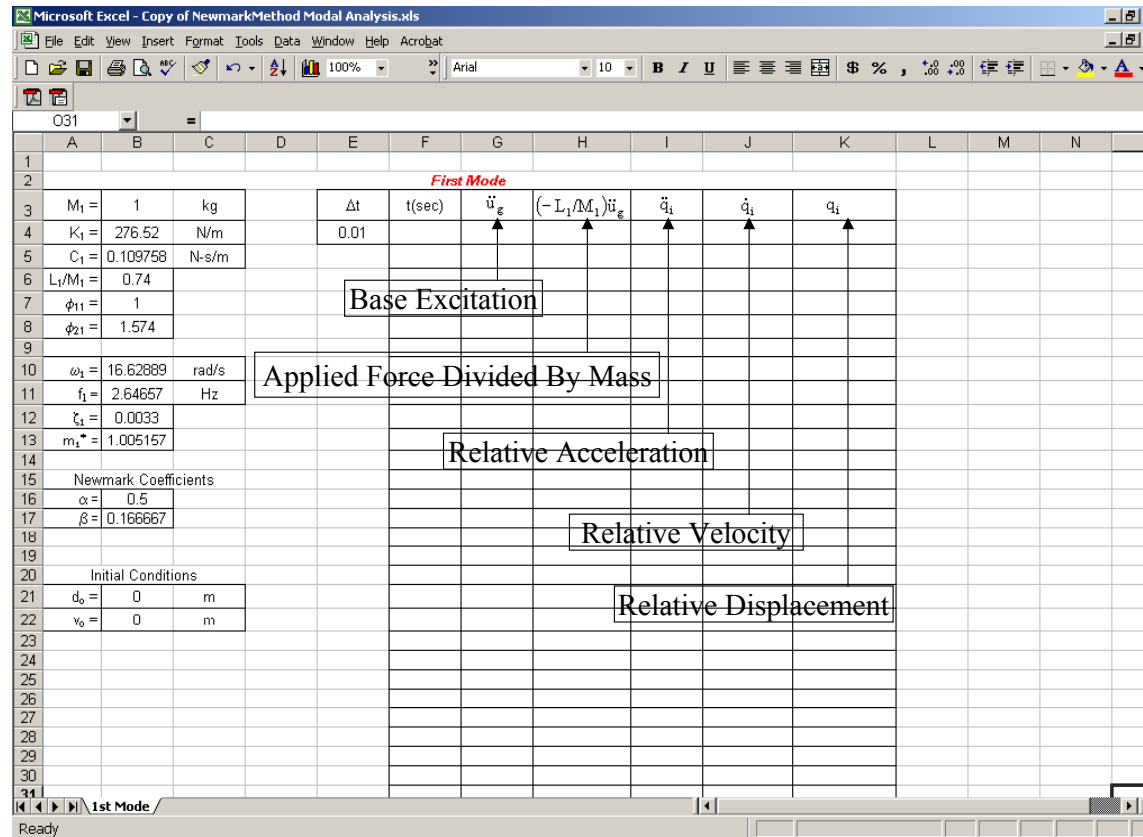


Figure 2: Spreadsheet After Completing Step 2

### *Step 3 – Enter the Time $t$ & Applied Force $f(t)$ into the Spreadsheet*

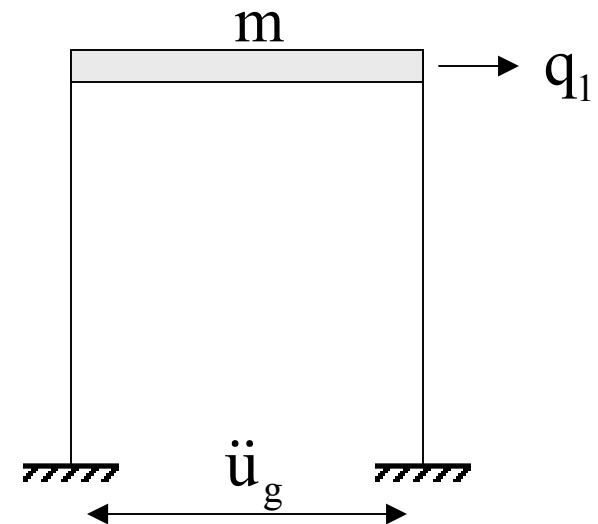
$$t_{i+1} = t_i + \Delta t \quad (\text{Equation 3}) \quad (\text{Fig. 3})$$

For the earthquake problem (acceleration applied to base of the structure), the applied force divided by the mass is calculated using:

$$\frac{f_i(t)}{M_1} = -\frac{L_1}{M_1} \ddot{u}_{g_i} \quad (\text{Equation 4}) \quad (\text{Fig. 3})$$

where,  $\ddot{u}_{g_i}$  is the applied base acceleration at step  $i$ . (Typically this is the base excitation time history)

**Check the units of the input motion file.  
They must be compatible with the units  
of the mass, stiffness, and damping!**



Microsoft Excel - Copy of NewmarkMethod Modal Analysis.xls

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H4 = -.\$B\$6\*G4

	A	B	C	D	E	F	G	H	I	J	K	L	M	N
1														
2														
3														
3	$M_1 =$	1	kg		$\Delta t$	$t(\text{sec})$	$\ddot{u}_g$	$(-L_1/M_1)\ddot{u}_g$	$\ddot{q}_1$	$\dot{q}_1$	$q_1$			
4	$K_1 =$	276.52	N/m		0.01	0	-0.06282	0.046483259						
5	$C_1 =$	0.109758	N-s/m		0.01	0.01	-0.05914	0.043764854						
6	$L_1/M_1 =$	0.74			0.02	0.02	0.005203	-0.003850502						
7	$\phi_{11} =$	1			0.03	0.03	0.075961	-0.056211422						
8	$\phi_{21} =$	1.574			0.04	0.04	0.067595	-0.050020003						
9					0.05	0.05	0.067458	-0.049919279						
10	$\omega_1 =$	16.62889	rad/s		0.06	0.06	0.065777	-0.048674691						
11	$f_1 =$	2.64657	Hz		0.07	0.07	0.063504	-0.046993152						
12	$\zeta_1 =$	0.0033			0.08	0.08	0.061549	-0.045545991						
13	$m_1^* =$	1.005157			0.09	0.09	0.060357	-0.044664359						
14					0.1	0.1	0.060173	-0.044528165						
3988					39.84	39.84	0.002425	-0.001794516						
3989					39.85	39.85	0.002226	-0.001646889						
3990					39.86	39.86	0.002042	-0.001511349						
3991					39.87	39.87	0.001873	-0.001385769						
3992					39.88	39.88	0.001723	-0.001274874						
3993					39.89	39.89	0.001598	-0.001182338						
3994					39.9	39.9	0.001496	-0.001106884						
3995					39.91	39.91	0.001411	-0.001044432						
3996					39.92	39.92	0.00134	-0.000991816						
3997					39.93	39.93	0.001281	-0.000947591						
3998					39.94	39.94	0.00123	-0.000910024						
3999					39.95	39.95	0.001183	-0.000875614						
4000					39.96	39.96	0.001134	-0.000838991						
4001					39.97	39.97	0.001075	-0.0007965						
4002					39.98	39.98	0.001006	-0.000744197						
4003					39.99	39.99	0.000928	-0.000686691						
4004														

Equation 3

Equation 4

Ready Sum=0.024046048

Figure 3: Spreadsheet After Completing Step 3

*Step 4 – Compute Initial Values of the Relative Acceleration, Relative Velocity, Relative Displacement, and Absolute Acceleration*

(A) The Initial Relative Displacement and Relative Velocity are known from the initial conditions (Fig. 4).

$$q(t = 0) = d_o \quad (\text{Equation 5})$$

$$\dot{q}(t = 0) = v_o \quad (\text{Equation 6})$$

(B) The Initial Relative Acceleration (Fig. 4) is calculated using

$$\ddot{q}(t = 0) = -\frac{L_i}{M_i} \ddot{u}_g - 2\zeta\omega v_o - \omega^2 d_o \quad (\text{Equation 7})$$

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	A	B	C	D	E	F	G	H	I	J	K	L	M	N
1														
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30														
31														

Ready

Figure 4: Spreadsheet After Completing Step 4

*Step 5 – Compute Incremental Values of the Relative Acceleration, Relative Velocity, Relative Displacement, and Absolute Acceleration At Each Time Step (Fig. 5)*

(A)

$$\ddot{q}_{i+1} = \frac{\left[ -\frac{L_1}{M_1} \ddot{u}_{g_{i+1}} - C_1 \left( \frac{\Delta t}{2} \ddot{q}_i + \dot{q}_i \right) - K_1 \left( \frac{1}{2} \Delta t^2 (1 - 2\beta) \ddot{q}_i + \Delta t \dot{q}_i + q_i \right) \right]}{m_1^*} \quad (\text{Equation 8})$$

$$\dot{q}_{i+1} = \ddot{q}_i \Delta t (1 - \alpha) + \ddot{q}_{i+1} \Delta t \alpha + \dot{q}_i \quad (\text{Equation 9})$$

$$q_{i+1} = \ddot{q}_i \frac{\Delta t^2}{2} (1 - 2\beta) + \ddot{q}_{i+1} \Delta t^2 \beta + \dot{q}_i \Delta t + q_i \quad (\text{Equation 10})$$

Where, the effective mass,  $m_1^* = M_1 + C_1 \Delta t \alpha + K_1 \Delta t^2 \beta$

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		First Mode						
		$\Delta t$	t(sec)	$\ddot{u}_g$	$(-L_1/M_1)\ddot{u}_g$	$\ddot{q}_1$	$\dot{q}_1$	$q_1$
3	$M_1 = 1$ kg	0.01	0	-0.06282	0.046483259	0.046483	0	0
4	$K_1 = 276.52$ N/m		0.01	-0.05914	0.043764854	0.043089	0.00044786	2.26759E-06
5	$C_1 = 0.109758$ N-s/m		0.02	0.005203	-0.003850502			
6	$L_1/M_1 = 0.74$		0.03	0.075961	-0.056211422			
7	$\phi_{11} = 1$		0.04	0.067595	-0.050020003			
8	$\phi_{21} = 1.574$		0.05	0.067458	-0.049919279			
9	$\omega_1 = 16.62889$ rad/s		0.06	0.065777	-0.048674691			
11	$f_1 = 2.64657$ Hz		0.07	0.063504	-0.046993152			
12	$\zeta_1 = 0.0033$		0.08	0.061549	-0.045545991			
13	$m_1^* = 1.005157$		0.09	0.060357	-0.044664359			
14			0.1	0.060173	-0.044528165			
15	Newmark Coefficients		0.11	0.060825	-0.045010552			
16	$\alpha = 0.5$		0.12	0.061601	-0.045584633			
17	$\beta = 0.166667$		0.13	0.061857	-0.045773878			
18			0.14	0.061563	-0.045556597			
19			0.15	0.06112	-0.045228799			
20	Initial Conditions		0.16	0.060828	-0.045012432			
21	$d_0 = 0$ m		0.17	0.060709	-0.044924986			
22	$v_0 = 0$ m		0.18	0.060653	-0.044800375			
23			0.19	0.060541	-0.0446800393			
24			0.2	0.060319	-0.044636076			
25			0.21	0.060005	-0.04440355			
26			0.22	0.059668	-0.044154408			
27			0.23	0.059424	-0.043973866			
28			0.24	0.059387	-0.043946302			
29			0.25	0.059559	-0.044073342			
30			0.26	0.059832	-0.04427556			
31			0.27	0.060157	-0.044516398			

Equation 10

Equation 9

Equation 8

1st Mode

Ready

Figure 5: Spreadsheet with values for the Relative Acceleration, Relative Velocity, and Relative Displacement at Time Step 1

(B) Then, highlight columns I, J, & K and rows 4 through to the last time step (in this example 4003) and “Fill Down” (Ctrl+D).

See Figures 6 and 7.

14003													
A	B	C	D	E	F	G	H	I	J	K	L	M	N
<i>First Mode</i>													
3	$M_1 =$	1	kg	$\Delta t$	$t(\text{sec})$	$\ddot{u}_g$	$(-L_1/M_1)\ddot{u}_g$	$\ddot{q}_i$	$\dot{q}_i$	$q_i$			
4	$K_1 =$	276.52	N/m	0.01	0	-0.06282	0.046483259	0.046483	0	0			
5	$C_1 =$	0.109758	N-s/m		0.01	-0.05914	0.043764854	0.043089	0.00044786	2.26759E-06			
6	$L_1/M_1 =$	0.74			0.02	0.005203	-0.003850502						
7	$\phi_{11} =$	1			0.03	0.075961	-0.056211422						
8	$\phi_{21} =$	1.574			0.04	0.067595	-0.050020003						
9					0.05	0.067459	-0.049919279						
10	$\omega_1 =$	16.62889	rad/s		0.06	0.065777	-0.048674691						
11	$f_1 =$	2.64657	Hz		0.07	0.063504	-0.046993152						
12	$\zeta_1 =$	0.0033			0.08	0.061549	-0.045545991						
13	$m_1^* =$	1.005157			0.09	0.060357	-0.044664359						
14					0.1	0.060173	-0.044528165						
15	Newmark Coefficients				0.11	0.060825	-0.045010552						
16	$\alpha =$	0.5			0.12	0.061601	-0.045584633						
17	$\beta =$	0.166667			0.13	0.061957	-0.045773878						
18					0.14	0.061963	-0.045556597						
19					0.15	0.06112	-0.045228799						
20	Initial Conditions				0.16	0.060828	-0.045012432						
21	$d_0 =$	0	m		0.17	0.060709	-0.044924986						
22	$v_0 =$	0	m		0.18	0.060653	-0.044800375						
23					0.19	0.060541	-0.044800393						
24					0.2	0.060319	-0.044636076						
25					0.21	0.060005	-0.04440355						
26					0.22	0.059668	-0.044154408						
27					0.23	0.059424	-0.043973866						
28					0.24	0.059387	-0.043946302						
29					0.25	0.059559	-0.044073342						
30					0.26	0.059832	-0.04427556						
31					0.27	0.060157	-0.044516398						

Figure 6: Highlighted Cells



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Formula Bar:  $= (H5 - \$B\$5 * (\$E\$4 / 2 * I4 + J4) - \$B\$4 * (\$E\$4 * 2 / 2 * (1 - 2 * \$B\$17) * I4 + \$E\$4 * J4 + K4)) / \$B\$13$

	A	B	C	D	E	F	G	H	I	J	K	L	M	N
1														
2														
3														
4														
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6														
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3991														
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3996														
3997														
3998														
3999														
4000														
4001														
4002														
4003														
4004														
4005														

Ready Sum=-10.36083541

Figure 7: Spreadsheet After “Filling Down” Columns I through K

## Step 6 – Create Additional Worksheet for Second Mode

Make a copy of the “1<sup>st</sup> Mode” worksheet by right clicking on the “1<sup>st</sup> Mode” tab and selecting “Move or Copy” (Fig. 8)

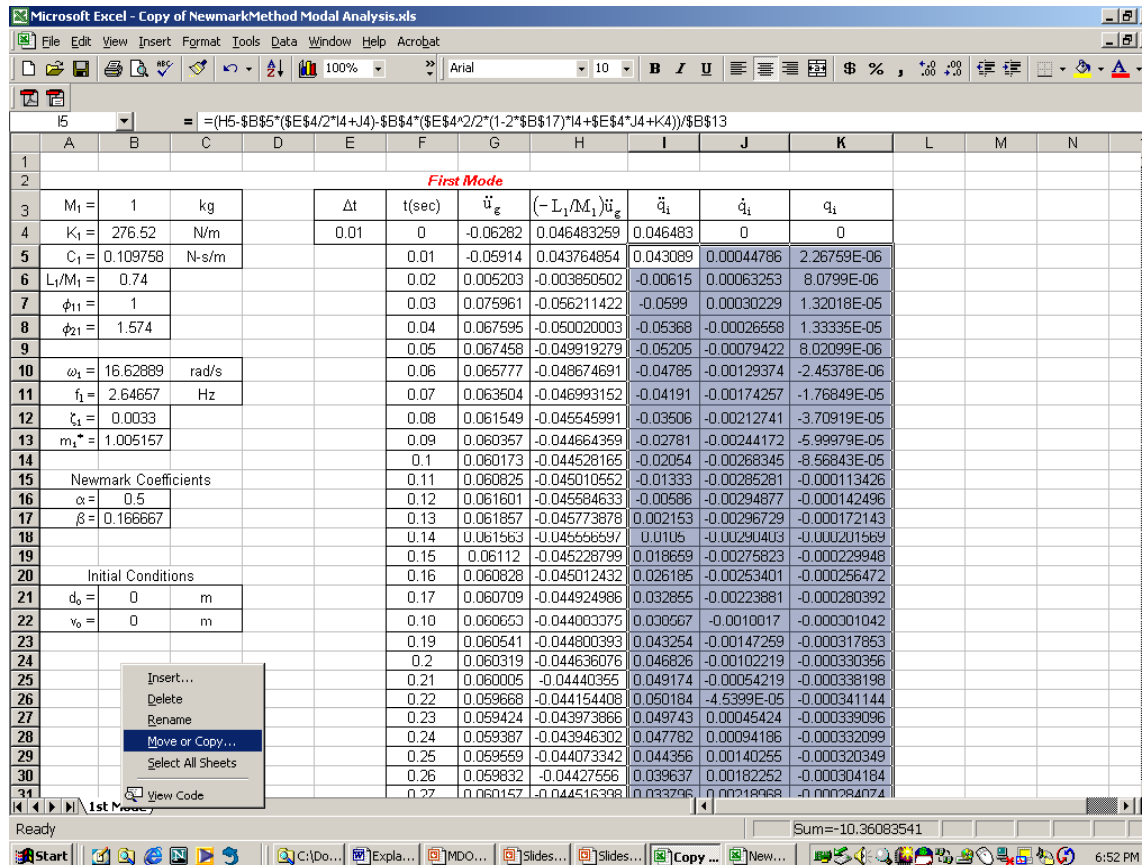


Figure 8: Creating a Copy of 1<sup>st</sup> Mode Worksheet

Then check the box for “Create a copy” and click on “OK” button (Fig. 9)

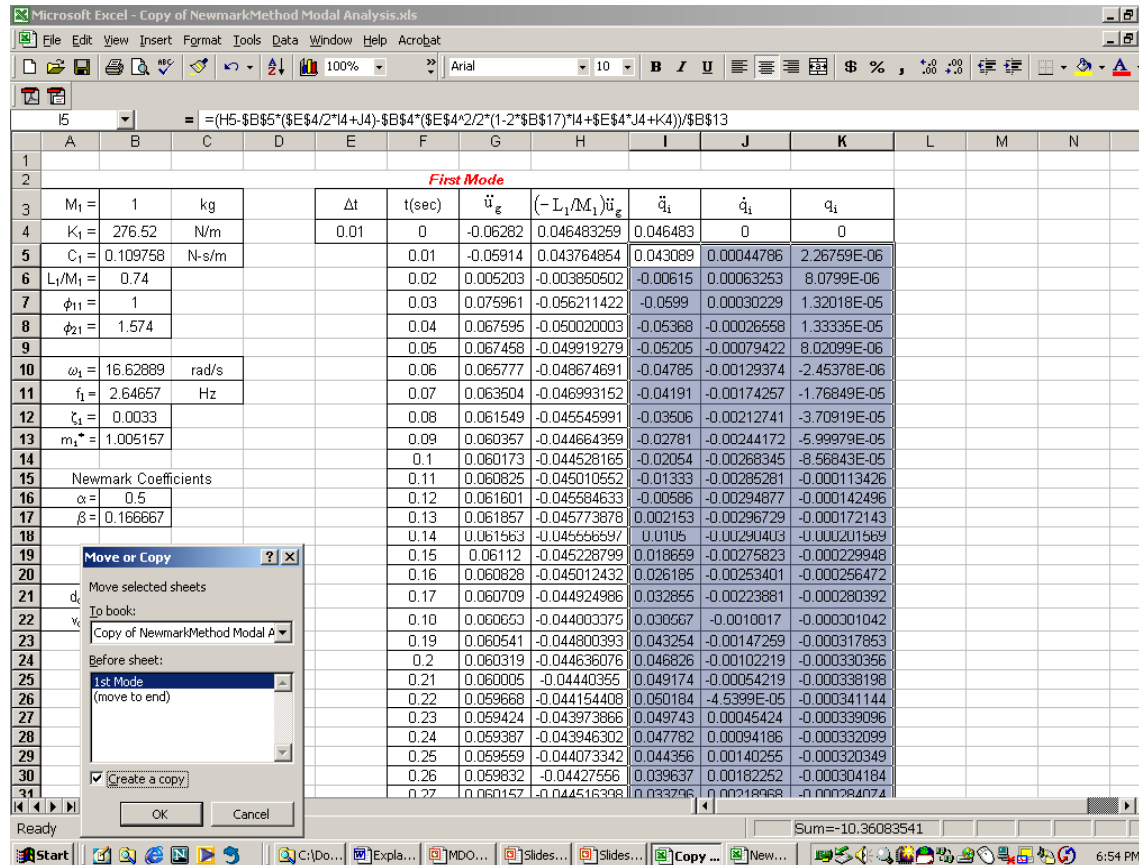


Figure 9: Creating a Copy of 1<sup>st</sup> Mode Worksheet

Rename this worksheet by right clicking on the “1st Mode (2)” tab and selecting “Rename”. Rename this worksheet “2<sup>nd</sup> Mode” (Fig. 10)

Enter the appropriate values for  $M_2$ ,  $K_2$ ,  $C_2$ ,  $\frac{L_2}{M_2}$ ,  $\phi_2$ ,  $d_o$ , and  $v_o$  (Fig. 10).

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	A	B	C	D	E	F	G	H	I	J	K	L	M	N
1														
2	<b>Second Mode</b>													
3	$M_2 =$	1	kg		$\Delta t$	t(sec)	$\ddot{u}_g$	$(-L_1/M_1)\ddot{u}_g$	$\ddot{q}_i$	$\dot{q}_i$	$q_i$			
4	$K_2 =$	1951.652	N/m		0.01	0	-0.06282	0.016331956	0.016332	0	0			
5	$C_2 =$	0.15152	N-s/m			0.01	-0.05914	0.01537684	0.013841	0.00015087	7.75087E-07			
6	$L_2/M_2 =$	0.26				0.02	0.005203	-0.001352879	-0.00653	0.00018744	2.63635E-06			
7	$\phi_{12} =$	1				0.03	0.075961	-0.019749959	-0.02725	1.8581E-05	3.83911E-06			
8	$\phi_{22} =$	-0.6356				0.04	0.067595	-0.017574596	-0.02288	-0.00023204	2.73543E-06			
9						0.05	0.067458	-0.017539206	-0.01627	-0.00042776	-6.1865E-07			
10	$\omega_2 =$	44.17751	rad/s			0.06	0.065777	-0.017101918	-0.0062	-0.00054012	-5.54191E-06			
11	$f_2 =$	7.031068	Hz			0.07	0.063504	-0.016511108	0.005164	-0.00054532	-1.10638E-05			
12	$\zeta_1 =$	0.001715				0.08	0.061549	-0.016002645	0.015461	-0.00044219	-1.60872E-05			
13	$m_2^* =$	1.033285				0.09	0.060357	-0.015692889	0.02263	-0.00025174	-1.96166E-05			
14						0.1	0.060173	-0.015645031	0.025261	-1.2284E-05	-2.09586E-05			
15	Newmark Coefficients					0.11	0.060825	-0.015814518	0.022906	0.00022855	-1.98576E-05			
16	$\alpha =$	0.5				0.12	0.061601	-0.016016222	0.016197	0.00042407	-1.65387E-05			
17	$\beta =$	0.166667				0.13	0.061857	-0.016082714	0.00657	0.0005379	-1.16486E-05			
18						0.14	0.061563	-0.016006372	-0.00415	0.00055002	-6.11969E-06			
19						0.15	0.06112	-0.01589912	-0.01403	0.00045916	-9.91467E-07			
20	Initial Conditions					0.16	0.060828	-0.015815179	-0.02128	0.00028263	2.77794E-06			
21	$d_0 =$	0	m			0.17	0.060709	-0.015784454	-0.02455	5.3498E-05	4.48582E-06			
22	$v_0 =$	0	m			0.18	0.060653	-0.015769034	-0.02319	-0.00010519	3.01607E-06			
23						0.19	0.060541	-0.015740678	-0.01744	-0.00038833	9.00572E-07			
24						0.2	0.060319	-0.015682946	-0.00838	-0.00051741	-3.70365E-06			
25						0.21	0.060005	-0.015601247	0.002279	-0.0005479	-9.11897E-06			
26						0.22	0.059668	-0.015513711	0.012494	-0.00047403	-1.43137E-05			
27						0.23	0.059424	-0.015450277	0.02031	-0.00031002	-1.82991E-05			
28						0.24	0.059387	-0.015440593	0.024227	-8.7328E-05	-2.03185E-05			
29						0.25	0.059559	-0.015486228	0.02351	0.00015136	-1.99924E-05			
30						0.26	0.059832	-0.015556278	0.018328	0.00036055	-1.73897E-05			
31						0.27	0.060157	-0.015640896	0.009678	0.00050057	-1.3012E-05			

Ready

Figure 10: Worksheet for Second Mode

*Step 7 – Repeat Step 6 for Additional Modes*

*Step 8 – Determine the Response at Each of the Floors*

Determine the Response of the first floor using the equations:

$$\mathbf{u} = \Phi \mathbf{q}$$

$$\dot{\mathbf{u}} = \Phi \dot{\mathbf{q}}$$

$$\ddot{\mathbf{u}} = \Phi \ddot{\mathbf{q}}$$

For example for a 2DOF structure, the first floor response is (Fig. 11)

$$u_1 = \phi_{11}q_1 + \phi_{12}q_2 \quad (\text{Equation 11})$$

$$\dot{u}_1 = \phi_{11}\dot{q}_1 + \phi_{12}\dot{q}_2 \quad (\text{Equation 12})$$

$$\ddot{u}_1 = \phi_{11}\ddot{q}_1 + \phi_{12}\ddot{q}_2 \quad (\text{Equation 13})$$

and the second floor response is (Fig. 12)

$$\mathbf{u}_2 = \boldsymbol{\phi}_{21}q_1 + \boldsymbol{\phi}_{22}q_2 \quad (\text{Equation 14})$$

$$\dot{\mathbf{u}}_2 = \boldsymbol{\phi}_{21}\dot{q}_1 + \boldsymbol{\phi}_{22}\dot{q}_2 \quad (\text{Equation 15})$$

$$\ddot{\mathbf{u}}_2 = \boldsymbol{\phi}_{21}\ddot{q}_1 + \boldsymbol{\phi}_{22}\ddot{q}_2 \quad (\text{Equation 16})$$

The first floor absolute acceleration is

$$\ddot{\mathbf{u}}_1^T = \ddot{\mathbf{u}}_1 + \ddot{\mathbf{u}}_g \quad (\text{Equation 17})$$

The second floor absolute acceleration is

$$\ddot{\mathbf{u}}_2^T = \ddot{\mathbf{u}}_2 + \ddot{\mathbf{u}}_g \quad (\text{Equation 18})$$



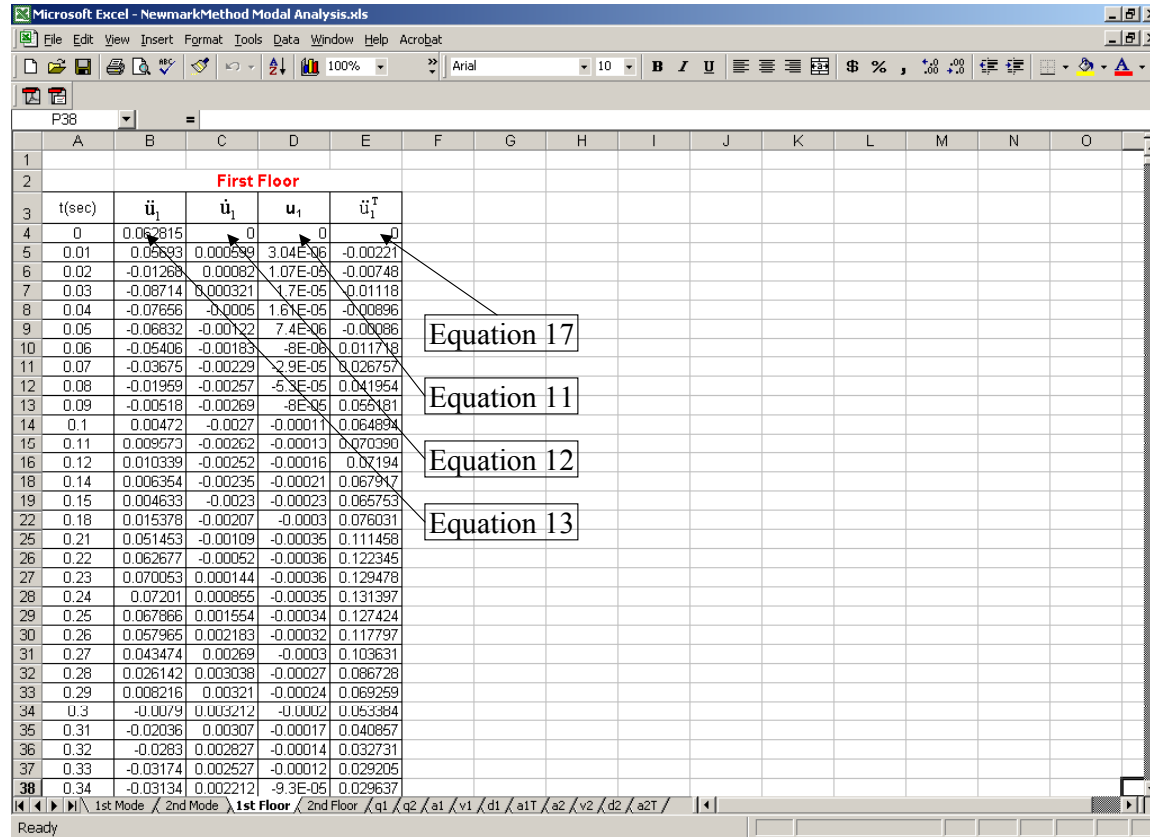


Figure 11: First Floor Response

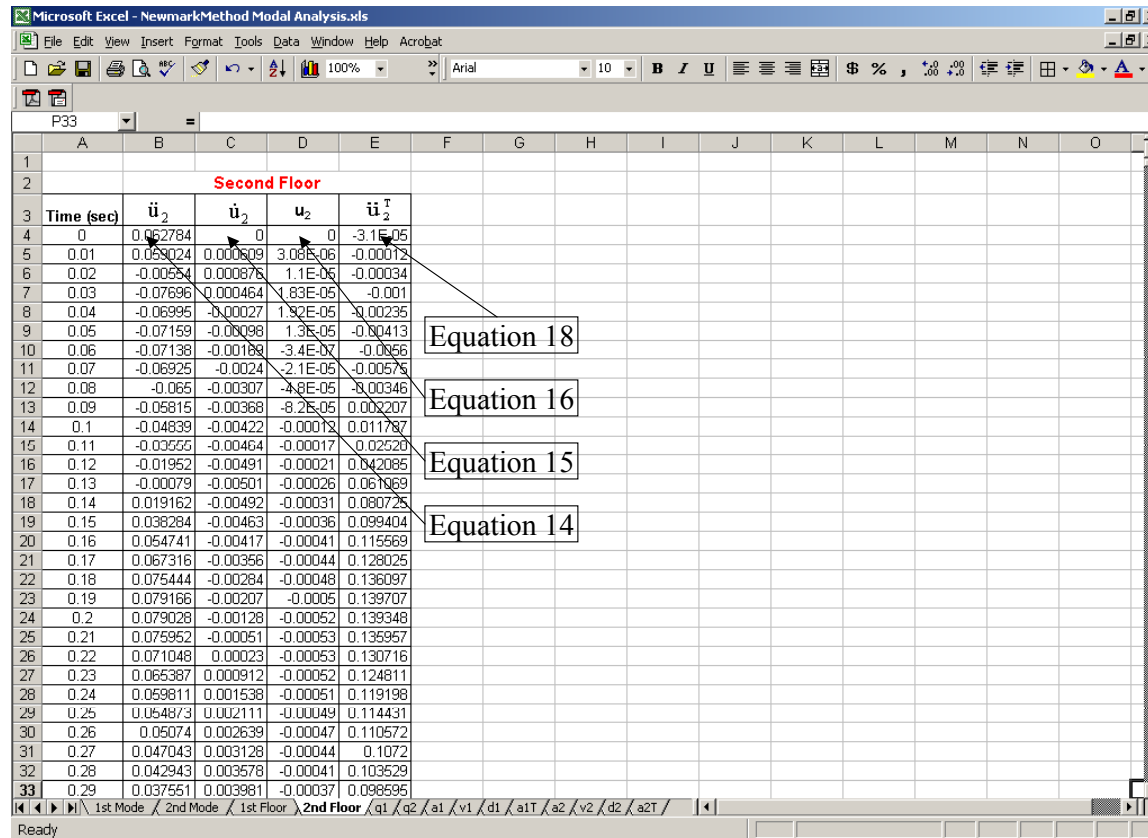


Figure 12: Second Floor Response